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Design and Optimization of Tidal Turbine Airfoil

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In order to increase the ratio of energy capture to the loading and thereby to reduce cost of energy, the use of specially tailored airfoils is needed. This work is focused on the design of an airfoil for marine application. Firstly, the requirements for this class of airfoils are illustrated and discussed with reference to the requirements for wind turbine airfoils. Then, the design approach is presented. This is a numerical optimization scheme in which a gradient based algorithm is used, coupled with RFOIL solver and a composite Bezier geometrical parameterization. A particularly sensitive point is the choice and implementation of constraints; in order to formalize in the most complete and effective way the design requirements, the effects of activating specific constraints are discussed. Particular importance is given to the cavitation phenomenon. Finally, a numerical example regarding the design of a high efficiency, tidal turbine airfoil is illustrated and the results are compared with existing turbine airfoils.

Nomenclature

α	=	angle of attack [deg]
c	=	airfoil chord [m]
C_d	=	airfoil drag coefficient [-]
C_{dmin}	=	minimum airfoil drag coefficient [-]
C_f	=	skin friction coefficient [-]
C_l	=	airfoil lift coefficient [-]
$C_{l\alpha}$	=	slope of the lift curve [deg ⁻¹]
C_{lmax}	=	maximum airfoil lift coefficient [-]
$C_{mc/4}$	=	airfoil moment coefficient referred to the quarter of chord [-]
c_p	=	pressure coefficient [-]
F	=	objective function
g	=	inequality constraints
h	=	equality constraints
H	=	boundary layer shape factor [-]
L/D	=	aerodynamic efficiency [-]
P_0	=	local pressure [Pa]
P_v	=	vapour pressure [Pa]
q	=	dynamic pressure [Pa]
σ_c	=	cavitation parameter [-]
X	=	design variables
X^L	=	lower bounds for the design variables
X^U	=	upper bounds for the design variables

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I. Introduction

DUE to the intrinsic requirements in terms of design point, off-design capabilities and structural properties, more and more, new airfoils families for wind turbines are developed [1-4]. However, apart from some development done by Eppler [5], there are not specific studies in literature focused on airfoils for tidal turbines.

This work is focused on the design of a tidal turbine dedicated airfoil, by using numerical optimization. In the next section, the requirements for this class of airfoils are presented, then the used design approach is explained. Finally, the development of the new airfoil is described and the results are discussed.

II. Airfoils for Tidal Turbines

In the present section, the requirements for tidal turbine airfoils are illustrated, by using wind turbine airfoil requirements as reference. A complete discussion for wind turbine airfoils can be found in [6].

A. Structural Requirements

Airfoil characteristics include both aerodynamic and structural requirements. For the outer part of the blade, the most important parameters, from the structural point of view, are the maximum airfoil thickness and the chord-wise location of the maximum thickness. The thickness of the profile must be able to accommodate the structure necessary for blade strength and stiffness. Depending of the class of the wind turbine, certain values for the thickness along the blade can be expected and this fact introduces a first indication for the design problem. The location of the maximum thickness along the chord is also important; when an airfoil is designed, also the other airfoils along the blade should be considered to guarantee constructive compatibility. This means that, in order to allow the spar passing through the blade, the chord-wise position of the thickness should be similar for the complete blade.

B. Aerodynamic Requirements

From the aerodynamic point of view the most important parameter for the tip region is the aerodynamic efficiency (L/D). In order to obtain good turbine performance, the aerodynamic efficiency should be as high as possible, but, at the same time, other considerations should be taken into account.

1. Cavitation

The biggest difference between tidal and wind turbine airfoils is connected with the cavitation phenomenon. Cavitation is the formation of vapor bubbles of a flowing liquid in a region where the pressure of the liquid falls below its vapor pressure. Cavitation is usually divided into two classes of behavior: inertial (or transient) cavitation,

and non-inertial cavitation. Inertial cavitation is the process where a void or bubble in a liquid rapidly collapses, producing a shock wave. Such cavitation often occurs in control valves, pumps, propellers, impellers, and in the vascular tissues of plants. Non-inertial cavitation is the process in which a bubble in a fluid is forced to oscillate in size or shape due to some form of energy input, such as an acoustic field. From the aerodynamic point of view, the inertial cavitation is undesirable because the shock waves formed are strong enough to significantly damage moving parts, so it should be taken into account during the design. The cavitation parameter (σ_c) is defined as:

$$\sigma_c = \frac{P_0 - P_v}{q} \quad (1)$$

Where, P_v is the vapour pressure, P_0 is the local pressure and q is the dynamic pressure. If the local pressure coefficient on the suction side is larger (in absolute value) than σ_c , then cavitation occurs.



Fig. 1 Cavitating propeller model in a water tunnel experiment.

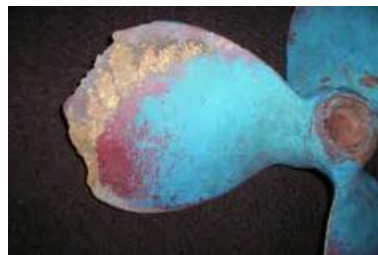


Fig. 2 Damage due to cavitation.

2. *Stall behavior and max lift coefficient*

Some of the existing airfoils for wind turbines have also a high value of the maximum lift coefficient (C_{lmax}) and a relative high value for the design lift coefficient (C_l); this means that, for a certain load, a smaller chord is

necessary. A lower chord in the outboard sections also reduces weight. For marine applications, the stall behavior is important, more than the C_{lmax} . So, the transition and the separation should move gradually when the angle of attack increases. A high C_l value (and lower associated chord c) reduces the amplitude of load fluctuations resulting from wind gusts and so fatigue loads. In water, the turbulence is lower; this means that problems connected with fatigue have lower priority. In wind turbines, because of gusts, the local angle of attack for the single airfoil can suddenly change and be in pre-stall or stall zone. So, it is important to have an angle of attack range between the design angle of attack and the one for which, noticeable separation occurs on the airfoil. For tidal turbines, especially if the turbine is a stall regulated turbine, it is convenient to reduce this margin to few degrees in order to use stall mechanism to stop the turbine as soon the turbine overcomes the design condition.

3. *Robustness to roughness*

Another important consideration is related with the sensitivity of the airfoil to the roughness. An airfoil with a large laminar flow extension will be very efficient in “clean” conditions, but very bad in case of “dirty” conditions. A large value for the leading edge can improve this aspect and, at the same time, it can help to avoid cavitation by preventing from rapid expansions.

4. *Blade torsion*

The moment coefficient (C_{m/c^4}) should be taken into account because large values of moment coefficient will give higher torsion moment on the blade. For tidal turbines however, the aspect ratio is lower than for wind turbines; this means that the blade is more rigid regarding torsion deformation, so the blade torsion does not play a crucial role in the design process.

III. Numerical Optimization Approach

In order to design airfoils, several methodologies can be used. A very popular approach is the inverse design technique, proposed by Lighthill and widely developed by Eppler [5-7] and Drela [8, 9]. The basic principle of this design method is that, the pressure coefficient on the airfoil surface is prescribed and the airfoil geometry is created; by iteratively modifying the pressure distribution on the airfoil surface, the designer can generate the geometry of an airfoil that satisfies the requirements. Despite its large use, there are several disadvantages associated with this technique; the most evident is that it is very difficult to take into account at the same time multiple requirements, especially when they concern different disciplines.

A valid alternative to solve this problem is the usage of multidisciplinary design optimization (MDO) approach. In the most general sense, numerical optimization [10, 11] solves the nonlinear, constrained problem to find the set of design variables, X_i , $i=1, N$, contained in vector \mathbf{X} , that will:

$$\text{Minimize } F(\mathbf{X}) \quad (2)$$

subject to:

$$g_j(\mathbf{X}) \leq 0 \quad j = 1, M \quad (3)$$

$$h_k(\mathbf{X}) = 0 \quad k = 1, L \quad (4)$$

$$X_i^L \leq X_i \leq X_i^U \quad i = 1, N \quad (5)$$

Equation 2 defines the objective function which depends on the values of the design variables, \mathbf{X} . Equations 3 and 4 are inequality and equality constraints respectively (equality constraints can be written as inequality constraints and included in equation 3), and equation 5 defines the region of search for the minimum. The bounds defined for each degree of freedom by equation 5 are referred to side constraints.

C. Geometry Description

One of the most important ingredients in numerical optimization is the choice of design variables and the parameterization of our system in using these variables. In order to reduce the number of necessary parameters to take into account to describe the airfoil's shape, but without loss of information about the geometrical characteristics of the airfoil, several mathematical formulations were proposed in literature [12]. In the present work, a composite third order Bezier is used. Basically, the airfoil is divided in four parts and for each part, a third order Bezier curve is used to describe the geometry. The advantage of this choice is the possibility to conjugate the properties of Bezier functions in terms of regularity of the curve and easy usage, with a piecewise structure that allows also local modifications to the geometry. The complete description can be found in [13]. a representative sketch is illustrated in Fig. 3; the points between 1 and 4 define the first Bezier curve, the ones between 4 and 7 the second curve and so on. In total, 13 control points are used to describe the airfoil geometry, that means 26 degrees of freedom. In practice, however, the leading edge is fixed (control point 7), as well as the abscissa of the trailing edge (control points 1 and 13); the abscissas of control points 6 and 8 are also fixed in order to maintain the curvature continuity at the leading edge and the control points 4 and 10 are directly controlled by the algorithm to keep the curvature continuity between two different Bezier curves.

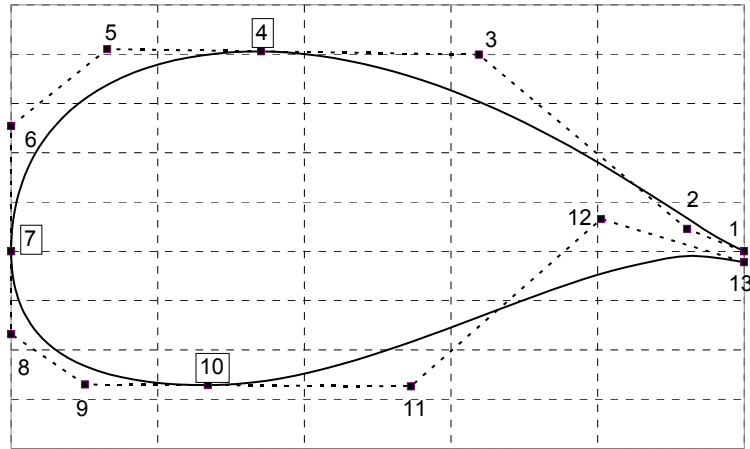


Fig. 3 Geometry parameterization example.

D. Optimization Algorithm

The choice of optimization algorithm is very important because the final results are usually dependent on the specific algorithm in terms of accuracy and local minima sensitivity. Evolutionary algorithms are less sensitive to local minima; however, they are time consuming and constraints have to be included as a penalty term to the objective function. On the other hand, gradient based algorithms can lack in global optimality but allow multiple constraints and are more robust, especially for problems in which a large number of constraints are prescribed. In this investigation, an advanced sequential quadratic programming (SQP) gradient based algorithm [14] is implemented and the gradients are approximated by finite differences.

E. Objective Function Evaluation

Since the optimization process requires many evaluations of the objective function and the constraints before an optimum design is obtained, the computational costs cannot be neglected, as well as the accuracy of the results. Here, the RFOIL [15] numerical code is used. RFOIL is a modified version of XFOIL [16] featuring an improved prediction around the maximum lift coefficient and capabilities of predicting the effect of rotation on airfoil characteristics. Regarding the maximum lift in particular, numerical stability improvements were obtained by using the Schlichting velocity profiles for the turbulent boundary layer, instead of Swafford's. Furthermore, the shear lag coefficient in Green's lag entrainment equation of the turbulent boundary layer model was adjusted and deviation from the equilibrium flow has been coupled to the shape factor of the boundary layer. The following figures

illustrate a comparison with experimental data [17] for the NACA-63,418 airfoil. The Reynolds number is 6 million and the transition is free.

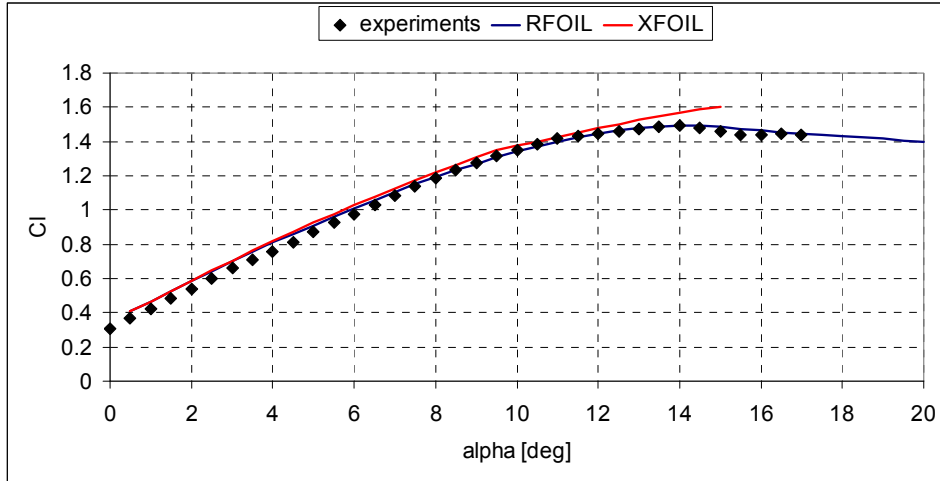


Fig. 4 Lift curve for the NACA-63,418 airfoil; comparison between XFOIL and RFOIL with experiments [17]. Reynolds number 6 million, free transition.

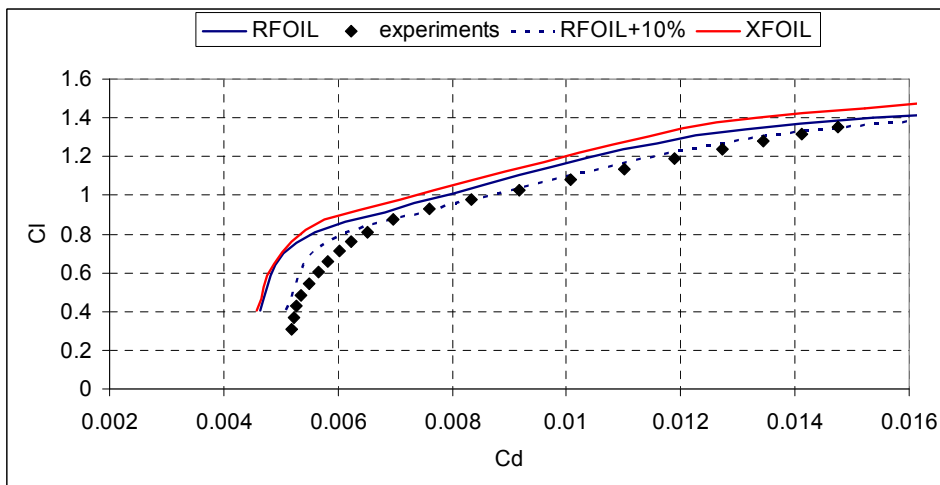


Fig. 5 Drag curve for the NACA-63,418 airfoil; comparison between XFOIL and RFOIL with experiments [17]. Reynolds number 6 million, free transition.

It should be noted that the RFOIL prediction for the stall region is well described and very close to the experimental data; in XFOIL results, only the deviation from the linear zone is described but not the stall. For the drag curve, XFOIL and RFOIL are very close to each other for small values of C_l , but for high C_l , XFOIL is under predicting. In [15], a additional drag of 10% is suggested to correct the RFOIL data; by adding this factor, a very

good agreement is found also for the drag coefficient. In order to have more realistic predictions, this 10% drag penalty is added during the optimization process and for all the numerical analyses.

IV. Design of a New Airfoil for Tidal Turbines

The design of a new airfoil is presented in this section. A stall regulated tidal turbine is chosen as reference; the Reynolds number is 3 million and the airfoil is designed to maximize the aerodynamic efficiency at 7 degrees of angle of attack. The same airfoil is used for all the blade.

The NACA0012 airfoil was used as baseline for the optimization. The purpose of this choice is to have a starting point for the design process, as far as possible from potential local solutions and, in this way, have more confidence on the optimality of the solution.

A. Geometrical Constraints

A minimum value of 18% for the airfoil thickness is prescribed. As consequence, it should be noted that, because of the 12% thickness, the baseline is even out of the feasible domain. The trailing edge thickness can change during the design, allowing the algorithm to find the best value; however, a minimum value of 0.25% of the chord is required in order to ensure airfoil's feasibility from manufacturing point of view.

One of the problems outlined in the previous sections is the insensitivity for the roughness and the need to have a smooth stall, with gradual transition and separation. By using the results of ESDU [18], a minimum value for the ordinate at x/c equal to $0.0125c$ can be selected to ensure a trailing edge separation. From this parameter, a minimum LE radius of $0.0155c$ can be assigned.

B. Aerodynamic Constraints

To avoid possibility of abrupt stall and converge to a solution in which a Stratford style recompression is not present (it can lead to a not gradual evolution in transition location), the design is performed by fixing transition at $0.01c$ on the suction side and $0.1c$ on the pressure side. However, to check the effect of the imposed transition on the final geometry, the optimization is performed also in free transition condition.

As mentioned before, preventing from cavitation is the main issue for marine applications. In the present work, the pressure coefficient (c_p) distribution is calculated for each geometry and the expansion peak is compared to the cavitation parameter. It is assumed a water temperature of 10°C , water depth equal to 1m and velocity of 10.5 m/s; from these values, a value equal to 2 for the cavitation parameter is calculated.

C. Results

Two airfoils have been designed: the G-hydra-A, obtained in free transition conditions and the G-hydra-B obtained in fixed transition conditions. Fig. 6 shows the comparison between these new airfoils, the NACA 4418 airfoil and the DU96-w-180 airfoil.

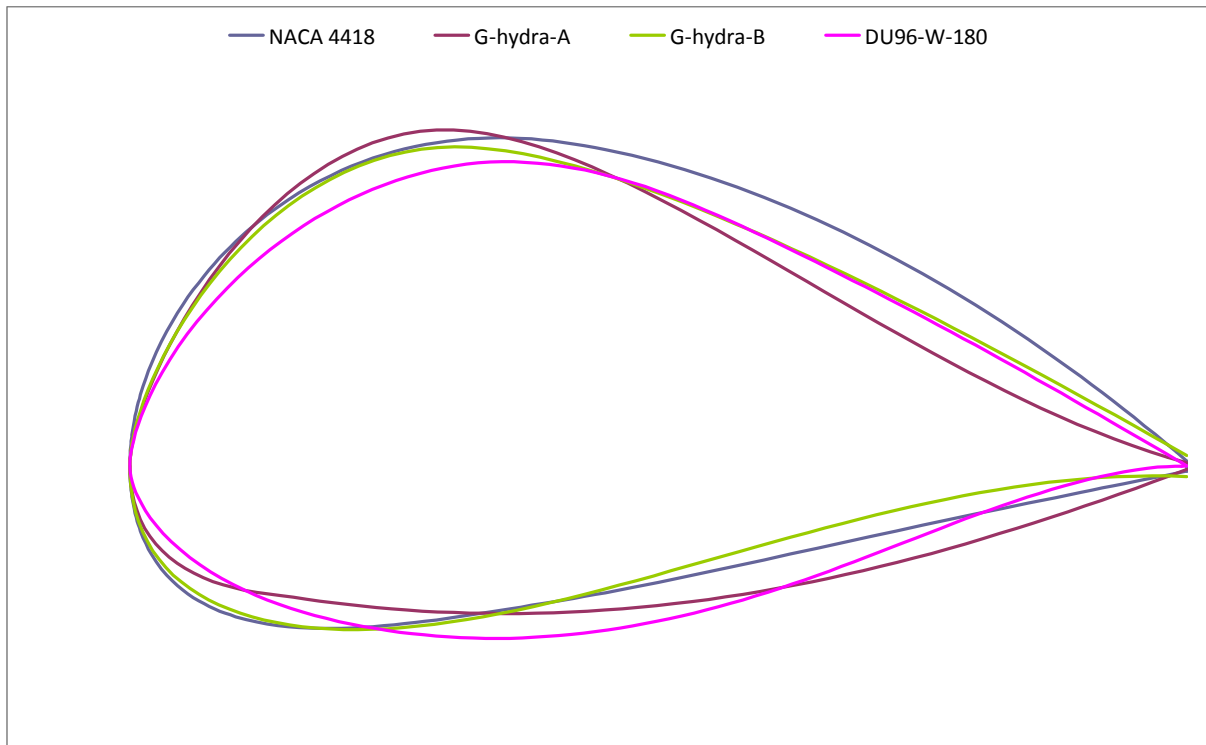


Fig. 6 G-hydra-A and G-hydra-B airfoils compared with NACA 4418 and DU96-W-180 geometries.

In Fig. 7 and Fig. 8, the aerodynamic characteristics are compared. Both airfoils improve the aerodynamic efficiency and respect the set of constraints. The main difference between G-hydra-A and G-hydra-B geometries is in the aft part of the airfoil. G-hydra-A airfoil is less cambered; because of this, the $C_{mc/4}$ (in absolute value) is lower, as well as the lift curve. Looking at the stall, the characteristics of the G-hydra-B airfoil are quite good in terms of quality of the stall and C_{lmax} . Due to the concave shape on the suction side of the G-hydra-A airfoil, the lift suddenly decreases as well as the efficiency, after the maximum is reached. Considering the G-hydra-B airfoil, there is some margin between design asset and stall, but not so long as for the NACA 4418. For stall regulated turbines, this can be an advantage because a little margin ensures that, from the design condition, the break mechanism (the

stall) starts to work before the turbine stay too long in off-design condition. Another good characteristic of the G-hydra-B airfoil is that the linear part of the lift curve is quite extended compared to the reference geometries.

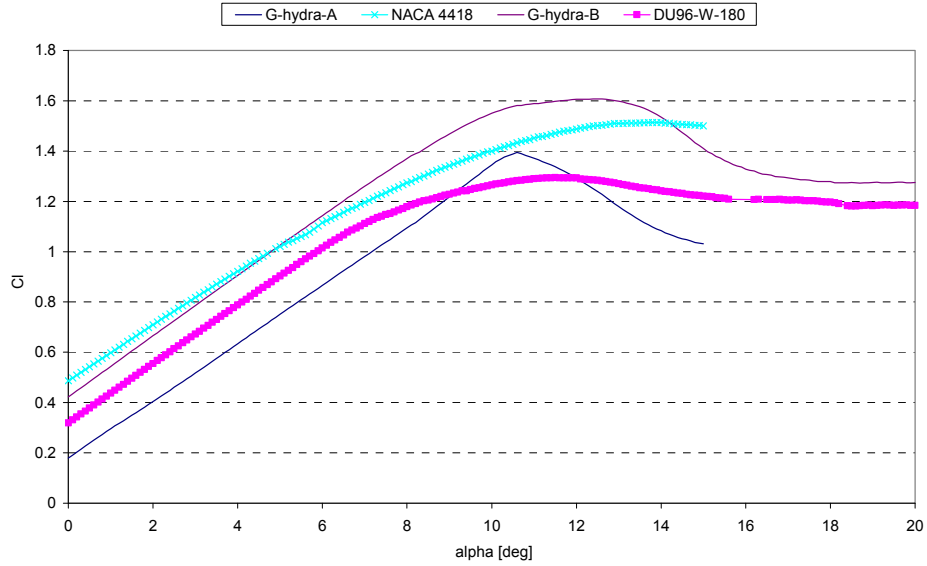


Fig. 7 Lift coefficient curve; G-hydra-A and G-hydra-B airfoils compared to NACA 4418 and DU96-W-180 geometries. Reynolds number 3 million, free transition, RFOIL predictions.

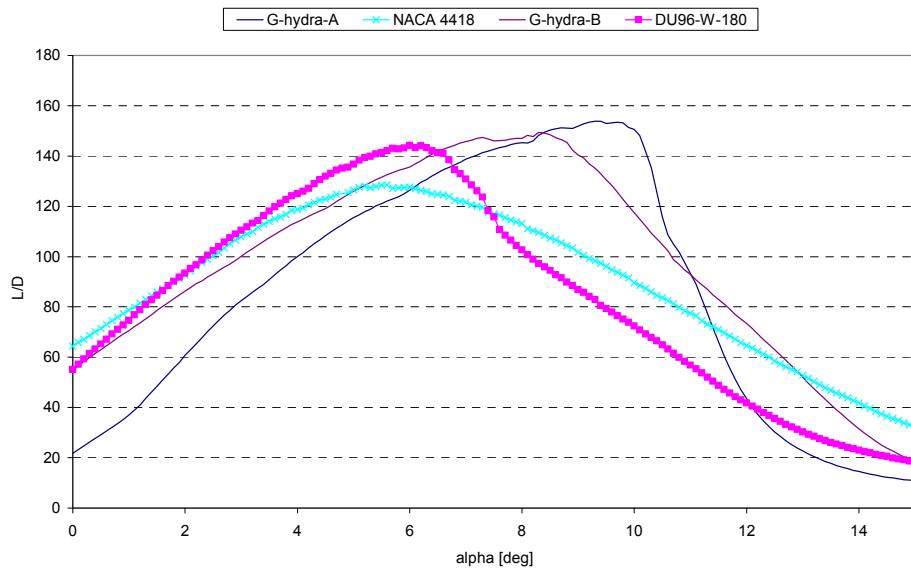


Fig. 8 Efficiency curve; G-hydra-A and G-hydra-B airfoils compared to NACA 4418 and DU96-W-180 geometries. Reynolds number 3 million, free transition, RFOIL predictions.

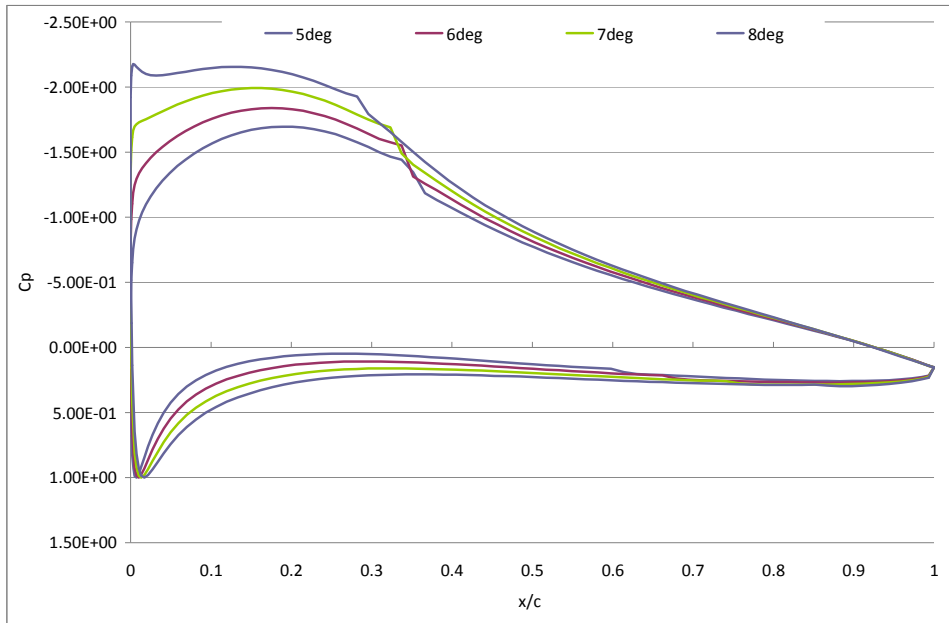


Fig. 9 Pressure coefficient distribution.

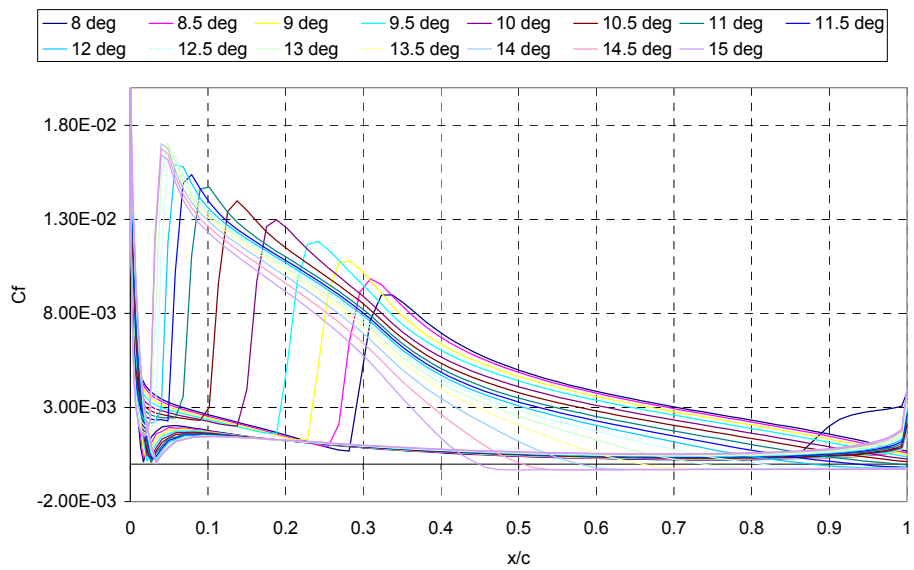


Fig. 10 C_f

The same comparison have been performed by imposing the location of the transition at 5% of the chord, both on the suction and the pressure sides.

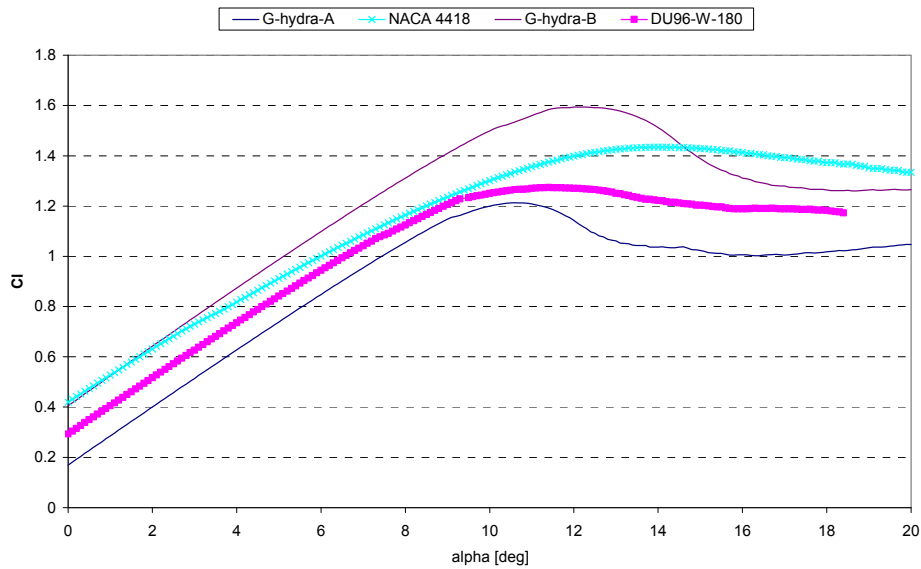


Fig. 11 Lift curve; G-hydra-A and G-hydra-B airfoils compared to NACA 4418 and DU96-W-180 geometries. Reynolds number 3 million, fixed transition, RFOIL predictions.

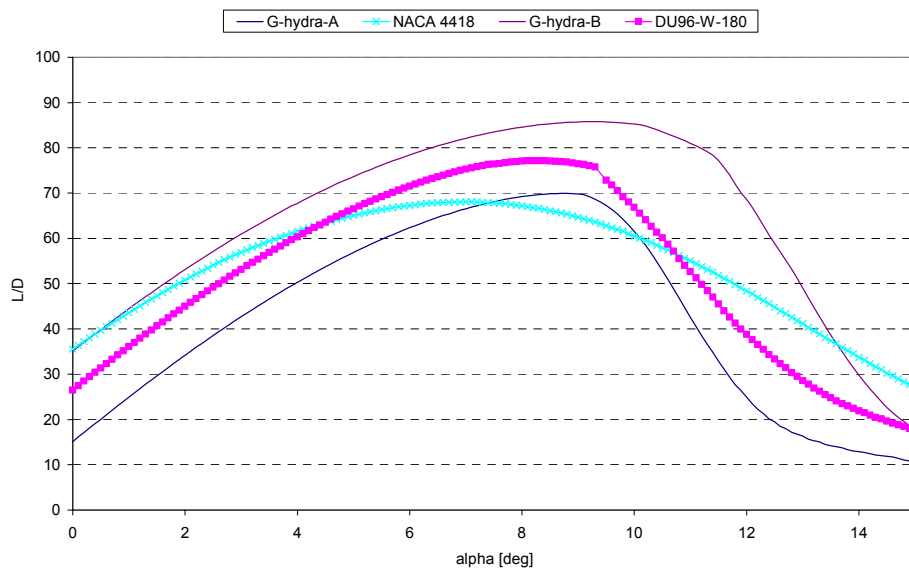


Fig. 12 Efficiency curve; G-hydra-A and G-hydra-B airfoils compared to NACA 4418 and DU96-W-180 geometries. Reynolds number 3 million, fixed transition, RFOIL predictions.

Fig. 11 shows the comparison in terms of lift curve. Looking at the extent of the linear part, the value of maximum lift coefficient and the quality of the stall, the lift curve of the G-hydra-B airfoil is almost unchanged in fixed transition conditions. Because of the imposed transition, the aerodynamic efficiency decreases for all the

airfoils (Fig. 12), however the loss of the G-hydra-B airfoil are limited, especially if compared to the G-hydra-A airfoil.

V. Conclusions

Two new airfoils for tidal turbines were designed. According to the RFOIL predictions, the results for one of them (G-hydra-B) are promising when compared to the NACA 4418 and the DU96-W-180 airfoils. A good value for the efficiency was achieved with separation limited only in the stall zone and without cavitation. Due to the relatively large leading edge radius, the performances in off-design and rough conditions are also good, as well as the stall that is quite smooth.

Despite these good results, wind tunnel tests are recommended to validate predictions. Especially for the stall behaviour, the numerical predictions, and consequently the MDO process used in this work, need to be verified. In absence of experimental data, the stall characteristics have been compared with the above mentioned work from ESDU; according to these data, the stall for the G-hydra-B airfoil is expected to be due to trailing edge separation.

Acknowledgments

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